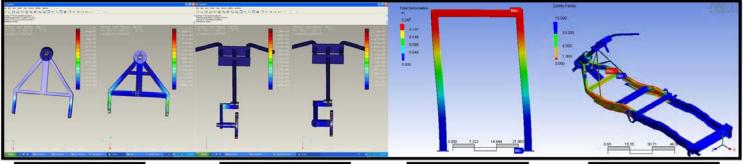
DESIGN ANALYSIS AND FABRICATI

Polaris Transmission Option

	· ·						
Low							
Engine Torque (ft-lbs)	Engine RPM	Eff. % CVT	CVT (ft-lbs)	CVT RPM	Trans. Ratio	Trans. RPM	Trans. Torque (ft-lbs)
13.7	3600	0.9	9.864	4050	4.5	810	39.9492
Eff. % Transmission	Diff. Torque (ft-lbs)	Diff. RPM	Tire Dia. (in)	Circumference (in)	Speed (mph)	Total Force (lbs)	Cost
0.9	143.81712	225	27.5	86.39379797	18.40776945	125.5131229	\$394.00
High							
Engine Torque (ft-lbs)	Engine RPM	Eff. % CVT	CVT (ft-lbs)	CVT RPM	Trans. Ratio	Trans. RPM	Trans. Torque (ft-lbs
13.7	3600	0.9	9.864	4050	7.49	486.6488652	66.493224
Eff. % Transmission	Diff. Torque (ft-lbs)	Diff. RPM	Tire Dia. (in)	Circumference (in)	Speed (mph)	Total Force (lbs)	Cost
0.9	239.3756064	135.1802403	27.5	86.39379797	11.05940755	208.9096201	\$394.00



frame under critical loading of 3000 lbs where the arrows are located.

Pro/E FEA Analysis of the Handle Bar assembly under critical loading of 3000 lbs where the arrows are

ANSYS FEA Analysis of the Roll Over Protective device under critical loading. This picture shows the deformation of the ROPS when a load is applied on the top right hand corner.

ANSYS FEA Analysis of the Frame under critical loading. This picture shows the Safety Factor of









The Formation of the Rear Frame



Pro/E Model of the BUV



Kyle welding the ROPS



Cory welding the front frame

